



ICE ACCRETION PREDICTION ON AN ENGINE INLET

Serkan Özgen ¹, Nermin Uğur ¹, İlhan Görgülü ², Volkan Tatar ²

¹ Middle East Technical University, Department of Aerospace Engineering

² TUSAŞ Engine Industries Inc.



Outline

- Motivation
- Methodology
- Results
- Conclusions
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Motivation

- In-flight icing on airframes and engines may cause great risk to flight safety due to **aerodynamic performance degradation and engine performance losses**.
- It is very important to simulate ice accretion to develop **ice protection systems** and to comply with **Airworthiness Requirements** (FAR/CS-25, App. C and very recently, App. O and P).
- A computational tool is developed for icing analyses in FORTRAN language within the scope of SANTEZ 0046.STZ.2013-1 Project.
- With this tool, **collection efficiencies and ice shape predictions** for a nacelle geometry are obtained in the present study. The results are compared with experimental and numerical data presented by Bidwell and Mohler [1], Iuliano et al [2].



Methodology

Ice Accretion Modeling Modules

1. Flow field solution
2. Droplet trajectories and collection efficiency calculations
3. Thermodynamic analyses
4. Ice accretion calculation



Methodology

1. Flow field solution

Hess-Smith Panel method

- The velocity and pressure distribution on the surface for boundary layer calculations
- Off-body velocities for droplet trajectory calculations

Methodology

1. Flow field solution

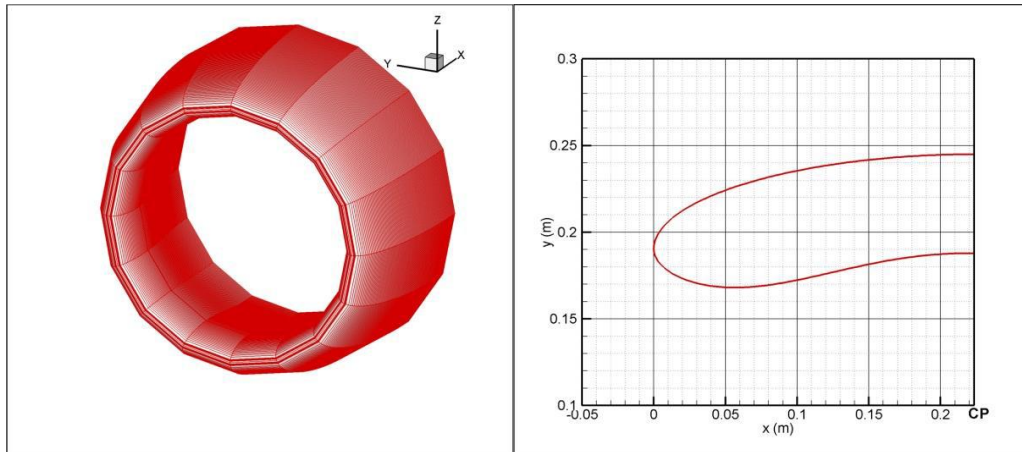


Figure 1: *Intake geometry*

- Outer and the inner cowls are defined by a super ellipse and an ellipse, respectively [2] .
- Length of the inlet: 0.2234 m
- Height of the inlet: 0.1905 m

Methodology

1. Flow field solution

Complicated to maintain both:

- the required flight conditions (freestream velocity)
- the desired mass flow rate through the intake

Superposition approach utilized by Waung [5]

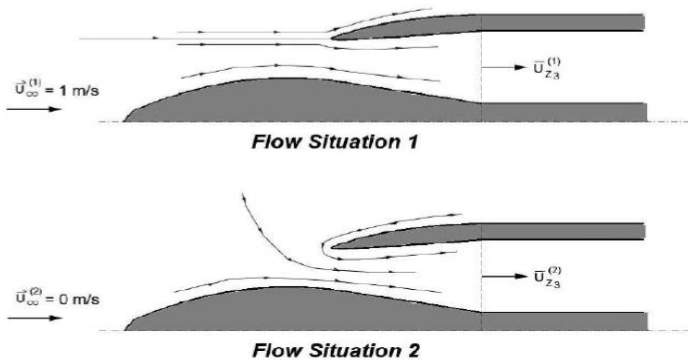


Figure 2: Visual representation of two flow situations used in the superposition method [5]

Flow situation 1: \rightarrow N+1 unknowns
 $U_\infty = 1 \text{ m/s}, \alpha = 0^\circ$

Flow situation 2: \rightarrow N unknowns
 $U_\infty = 0 \text{ m/s}, \Gamma = 1$
 (Γ : vortex strength along the surface panels)

- The final flow is obtained by scaling and combining these two solutions.

$$\begin{aligned} c_1 U_{\infty 1} + c_2 U_{\infty 2} &= U_\infty, \\ c_1 \bar{U}_{cp1} + c_2 \bar{U}_{cp2} &= \bar{U}_{cp}. \end{aligned}$$

$$c_1 = U_\infty \text{ and } c_2 = (\bar{U}_{cp} - U_\infty \bar{U}_{cp1}) / \bar{U}_{cp2}.$$

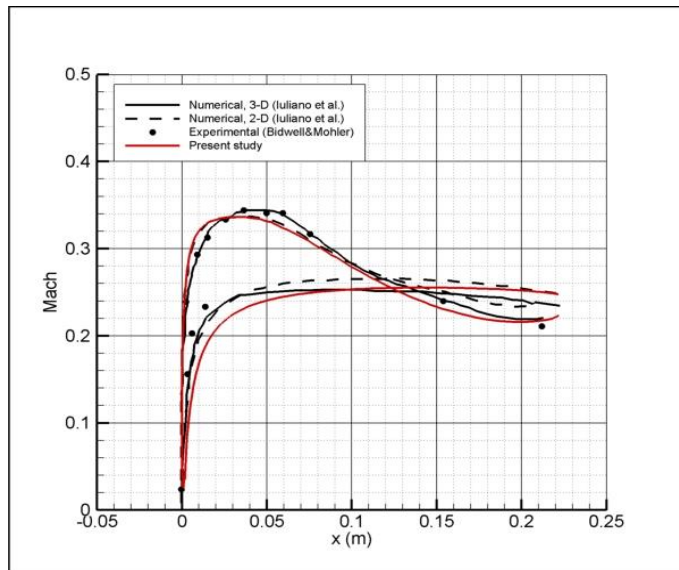
The velocity components are corrected for compressibility effects using the Prandtl-Glauert compressibility correction:

$$\hat{u} = \bar{u} / \sqrt{1 - M^2}, \quad \hat{v} = \bar{v} / \sqrt{1 - M^2},$$

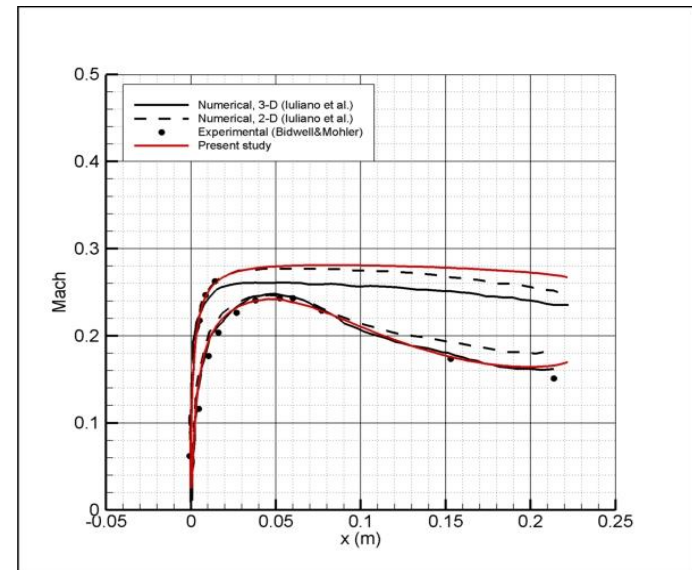
Methodology

1. Flow field solution

V_∞ (m/s)	α (degree)
75	0



a) $\dot{m} = 10.42$ kg/s



b) $\dot{m} = 7.8$ kg/s

Figure 3: *Mach number distributions on the intake.*

Methodology

2. Calculation of droplet trajectories

Lagrangian approach

$$m\ddot{x}_p = -D \cos \gamma,$$

$$m\ddot{y}_p = -D \cos \gamma + mg,$$

$$\gamma = \tan^{-1} \frac{\dot{y}_p - V_y}{\dot{x}_p - V_x},$$

$$D = \frac{1}{2} \rho V_{rel}^2 C_D A_p,$$

$$V_{rel} = \sqrt{(\dot{x}_p - V_x)^2 + (\dot{y}_p - V_y)^2}.$$

Collection efficiency (β)

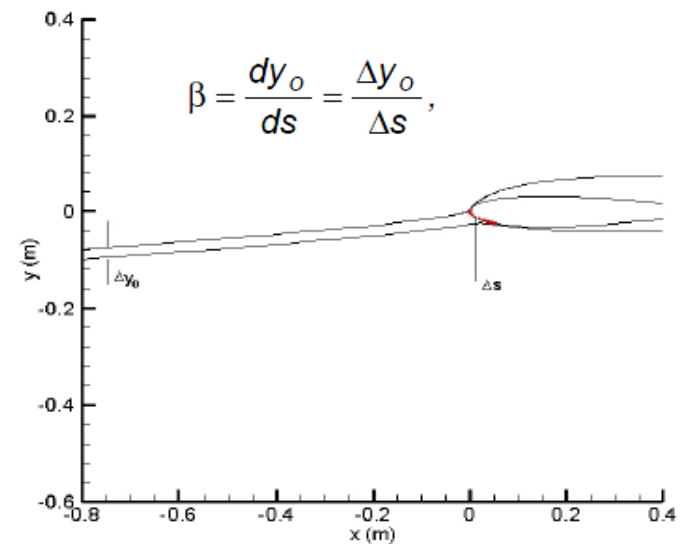


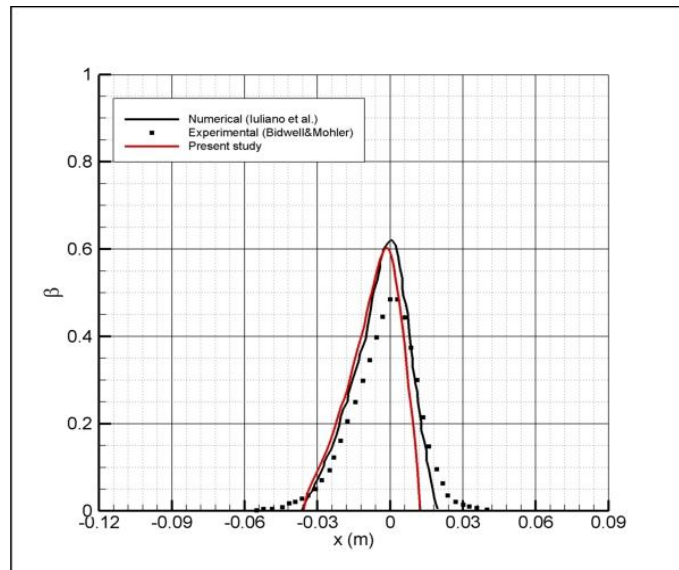
Figure 4: *Definition of collection efficiency*

Methodology

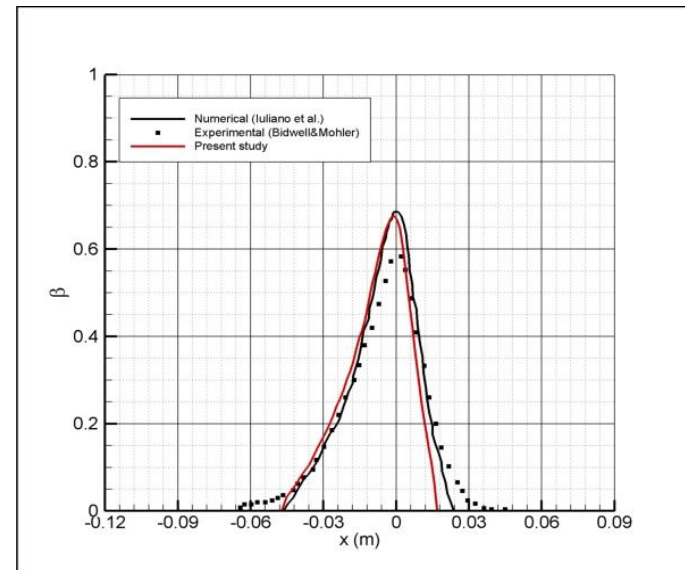
2. Calculation of droplet trajectories

Collection efficiency results 1

\dot{m} (kg/s)	α (degree)
10.42	0



a) $d_p = 16.45$ microns



b) $d_p = 20.36$ microns

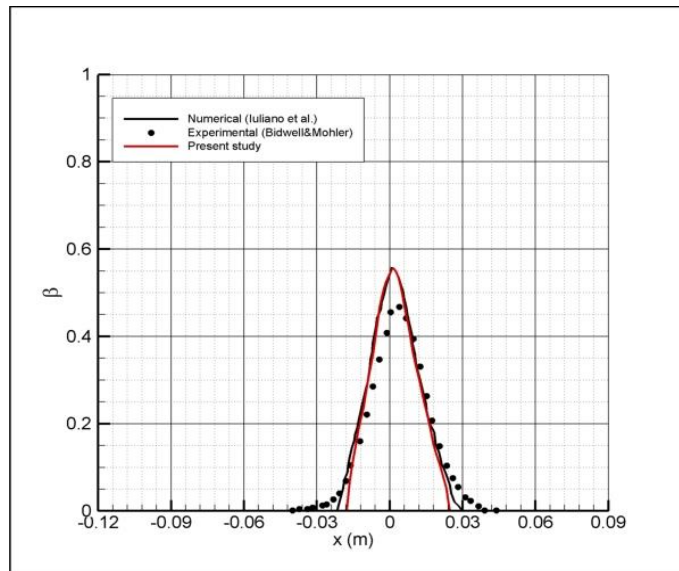
Figure 5: Collection efficiency distribution on the nacelle

Methodology

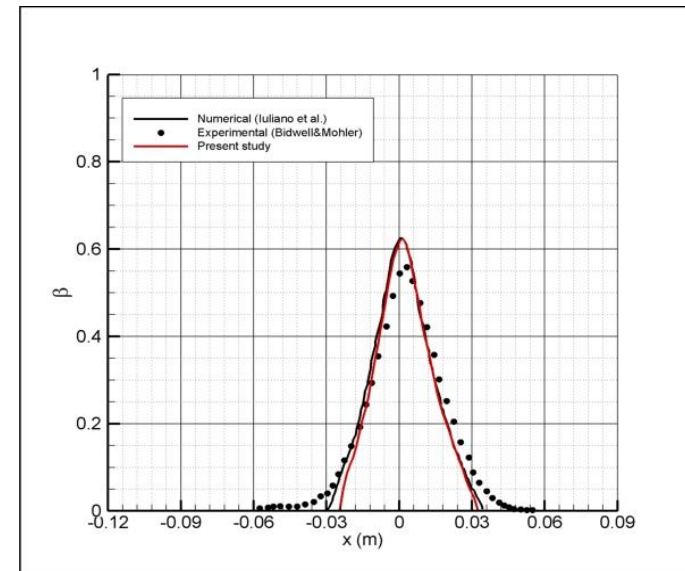
2. Calculation of droplet trajectories

Collection efficiency results 2

\dot{m} (kg/s)	α (degree)
7.8	0



a) $d_p = 16.45$ microns



b) $d_p = 20.36$ microns

Figure 6: Collection efficiency distribution on the nacelle



Methodology

3. Thermodynamic analyses

Calculation of convective heat transfer coefficients:

➤ 2D Integral Boundary Layer Method

Laminar

Turbulent

➤ Smith and Spaulding

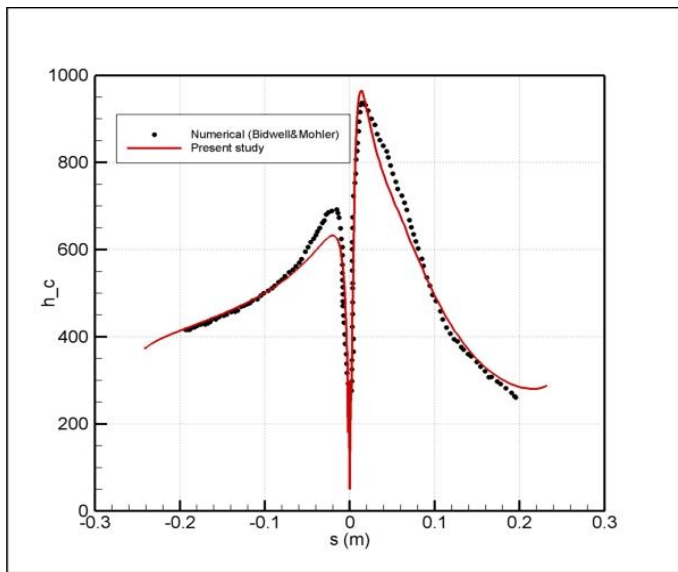
➤ Kays and Crawford

▪ Transition from laminar to turbulent flow occurs when the Reynolds number based on roughness height exceeds $Re_k=600$

where $Re_k = \rho U_k k_s / \mu$

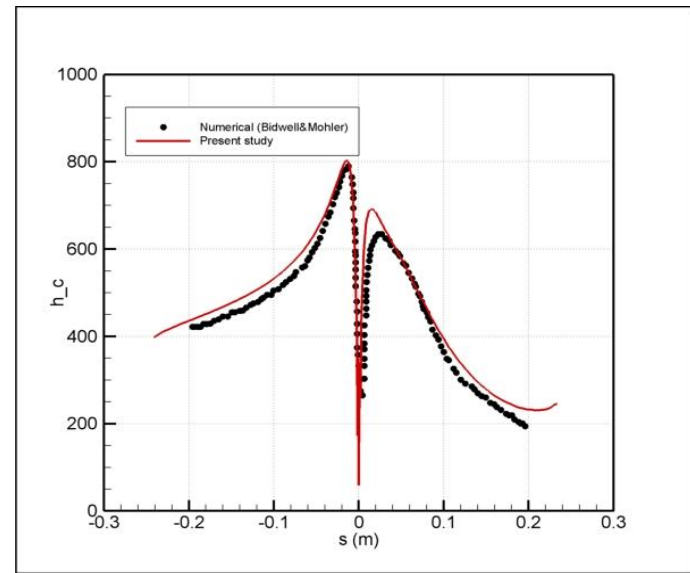
Methodology

3. Thermodynamic analyses



$\dot{m} = 10.42 \text{ kg/s}$

V_∞ (m/s)	α (degree)
75	0



$\dot{m} = 7.8 \text{ kg/s}$

Figure 7: *The heat transfer coefficient distribution on the nacelle*



Methodology

4. Ice accretion calculation

- Extended Messinger Model [3]

Stefan problem

Energy equation for ice layer:

$$\frac{\partial T}{\partial t} = \frac{k_i}{\rho_i c_{pi}} \frac{\partial^2 T}{\partial y^2} ,$$

Energy equation for water layer:

$$\frac{\partial \theta}{\partial t} = \frac{k_w}{\rho_w c_{pw}} \frac{\partial^2 \theta}{\partial y^2} ,$$

Mass balance:

$$\rho_i \frac{\partial B}{\partial t} + \rho_w \frac{\partial h}{\partial t} = \rho_a \beta V_\infty + \dot{m}_{in} - \dot{m}_{e,s} ,$$

Phase change condition:

$$\rho_i L_F \frac{\partial B}{\partial t} = k_i \frac{\partial T}{\partial y} - k_w \frac{\partial \theta}{\partial y} ,$$



Methodology

4. Ice accretion calculation

Boundary and initial conditions for Stefan problem:

1. $T(0,t) = T_s = T_a$
2. $T(B,t) = \theta(B,t) = T_f$
3. $B = h = 0$ at $t = 0$
4. At the air/water (glaze ice) or air/ice (rime ice) interface, heat flux is determined by convection (Q_c), radiation (Q_r), latent heat release (Q_l), cooling by incoming droplets (Q_d), heat brought in by runback water (Q_{in}), evaporation (Q_e) or sublimation (Q_s), aerodynamic heating (Q_a) and kinetic energy of incoming droplets (Q_k).

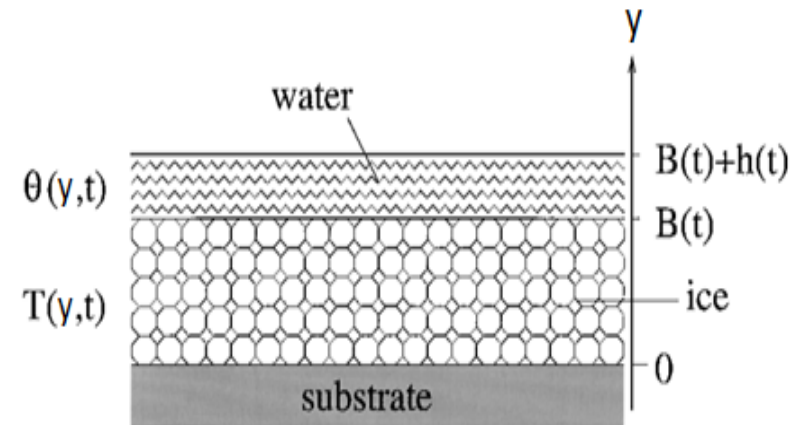


Figure 8: Water and ice layers on a surface [3]



Methodology

4. Ice accretion calculation

- Rime ice growth :
$$B(t) = \frac{\rho_a \beta V_\infty}{\rho_r} t$$

(Algebraic equation)

- Glaze ice growth :
$$\rho_g L_F \frac{\partial B}{\partial t} = \frac{k_i (T_f - T_s)}{B} + k_w \frac{(Q_c + Q_e + Q_d + Q_r) - (Q_a + Q_k)}{k_w}$$

(1st order ODE)

- The equations are integrated over time using a variable stepsize Runge-Kutta integrator.



Results

Ice shape predictions

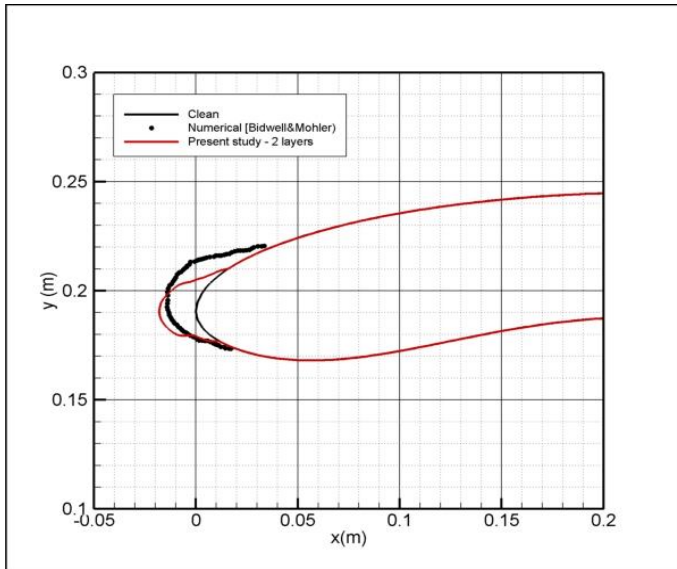
Table 1: *Icing conditions* ($V_\infty=75$ m/s, $\alpha=0^\circ$, $t_{exp}=30$ min)

Case #	\dot{m} (kg/s)	d_p (microns)	ρ_a (g/m ³)	T_a (°C)	Condition
1	10.42	16.45	0.2	-29.9	Rime
2	10.42	20.36	0.2	-29.9	Rime
3	7.8	16.45	0.2	-29.9	Rime
4	7.8	20.36	0.2	-29.9	Rime
5	10.42	16.45	0.695	-9.3	Glaze
6	10.42	20.36	0.695	-9.3	Glaze
7	7.8	16.45	0.695	-9.3	Glaze
8	7.8	20.36	0.695	-9.3	Glaze

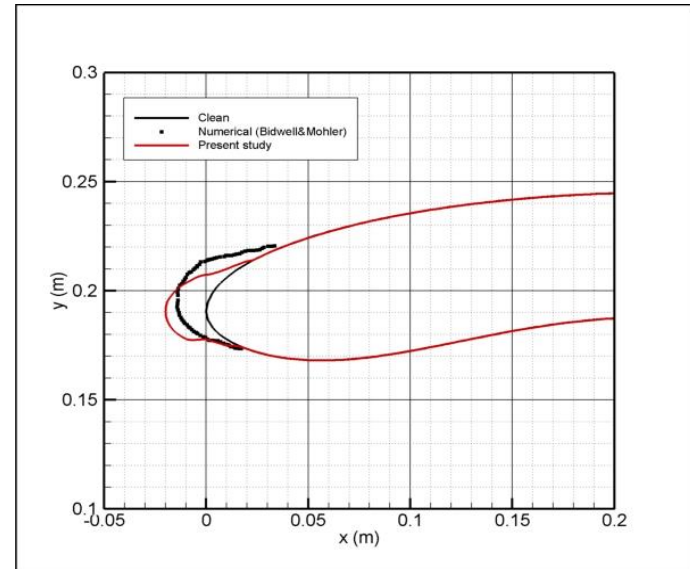
Results

Rime ice condition 1

\dot{m} (kg/s)	ρ_a (g/m ³)	T_a (°C)
10.42	0.2	-29.9



a) $d_p = 16.45$ microns



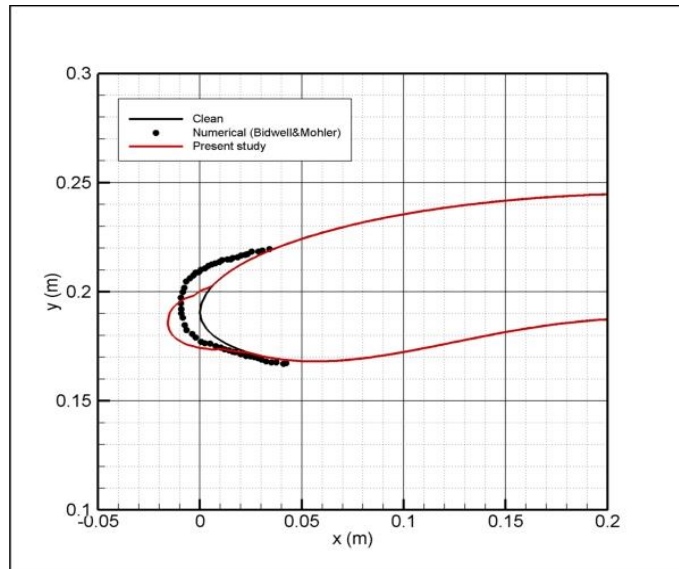
b) $d_p = 20.36$ microns

Figure 9: Ice shapes on the air intake

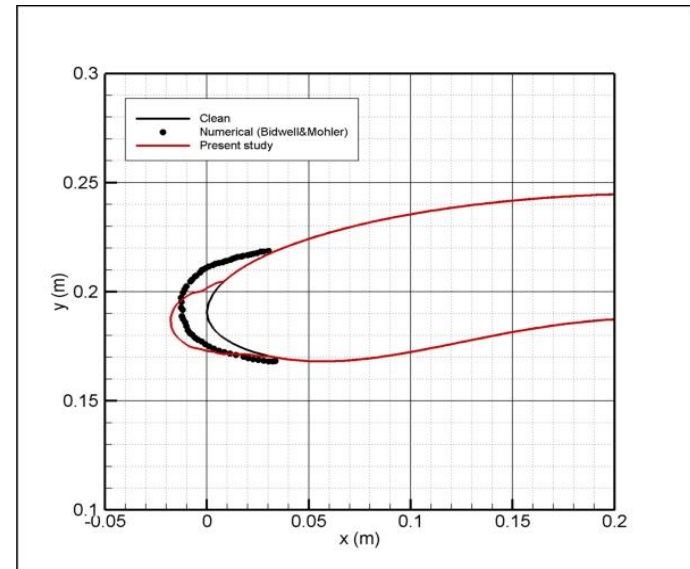
Results

Rime ice condition 2

\dot{m} (kg/s)	ρ_a (g/m ³)	T_a (°C)
7.8	0.2	-29.9



a) $d_p = 16.45$ microns



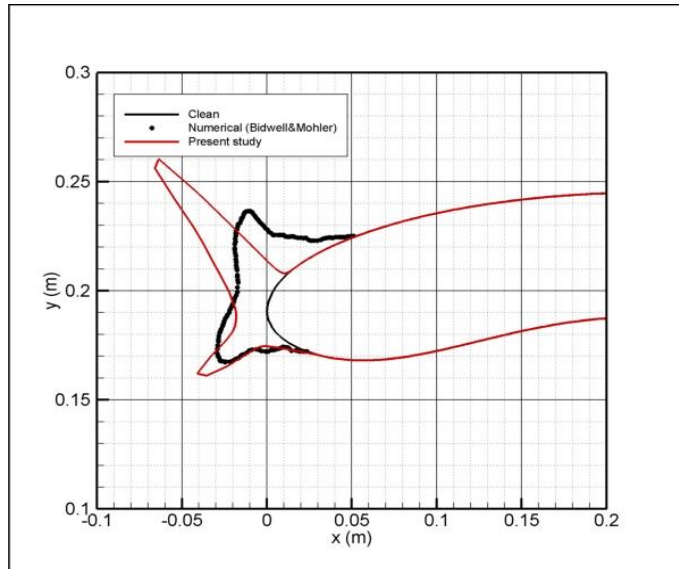
b) $d_p = 20.36$ microns

Figure 10: Ice shapes on the air intake

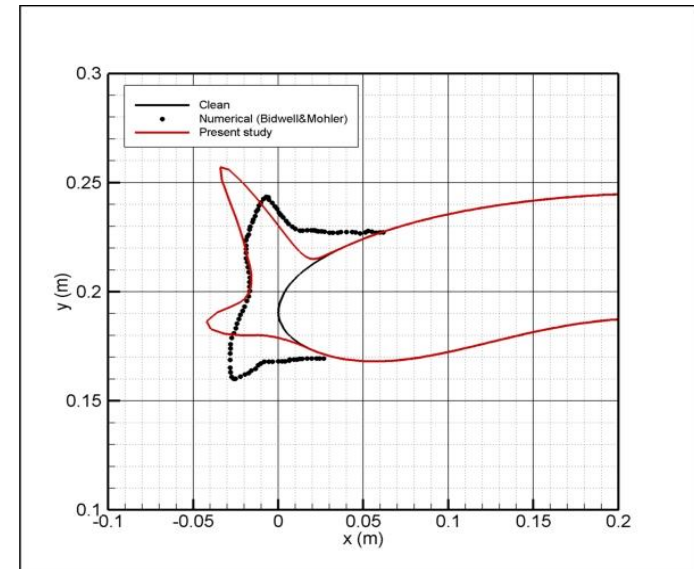
Results

Glaze ice condition 1

\dot{m} (kg/s)	ρ_a (g/m ³)	T_a (°C)
10.42	0.695	-9.3



a) $d_p = 16.45$ microns



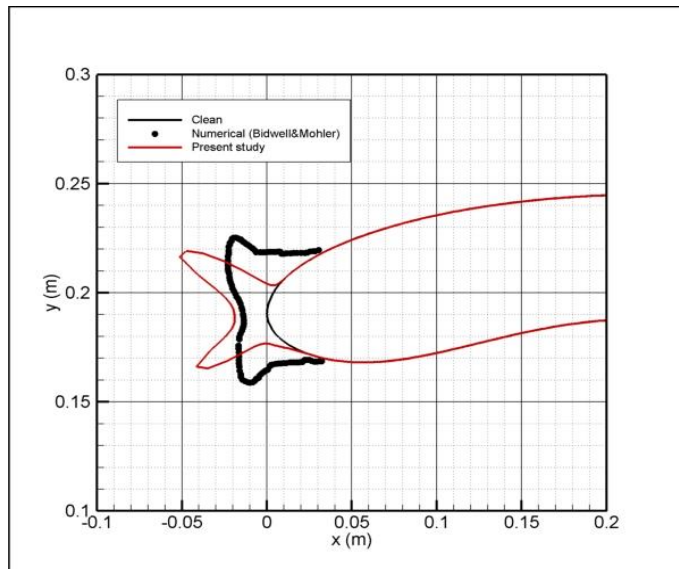
b) $d_p = 20.36$ microns

Figure 11: Ice shapes on the air intake

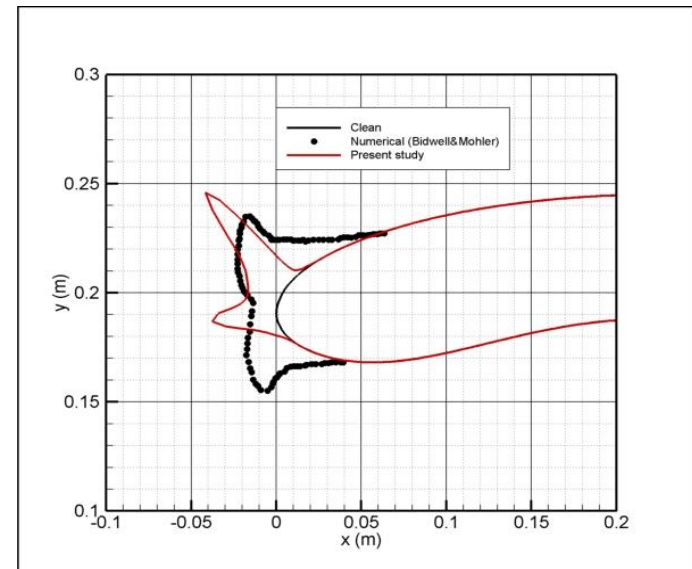
Results

Glaze ice condition 2

\dot{m} (kg/s)	ρ_a (g/m ³)	T_a (°C)
7.8	0.695	-9.3



a) $d_p = 16.45$ microns



b) $d_p = 20.36$ microns

Figure 12: Ice shapes on the air intake



Conclusions

- Collection efficiency and ice shape results are presented for a benchmark intake geometry in symmetrical flow conditions.
- It is concluded that larger droplets results in wider impingement zones and higher collection efficiencies.
- In other words, larger droplets lead to larger and thicker ice formations.
- It is also verified that the proposed method for maintaining the mass flow rate through the inlet is valid.



Acknowledgement

- Ministry of Science, Industry and Technology.
- TUSAŞ Engine Industries Inc.





References

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- [3] Myers, T.G., Extension to the Messinger Model for aircraft icing, *AIAA J.*, Vol. 39, pp. 211-218, 2001.

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- [5] Waung, T.S., An Ejector Inlet Design Method for a Novel Rocket-Based Combined_cycle Rocket Nozzle, M.Sc. Thesis, Ottawa-Carlton Institute for Mechanical and Aerospace Engineering, 2010.